

1 SEALED IMPELLER FOR PRODUCING METAL FOAM
2 AND SYSTEM AND METHOD THEREFOR

3
4 BACKGROUND OF THE INVENTION5
6 FIELD OF THE INVENTION

7 [0001] The present invention relates generally to submerged impellers and, more
8 particularly, to impellers used in generating metal foam.

9
10 DESCRIPTION OF THE PRIOR ART

11 [0002] There is a considerable demand for materials having high strength and low weight
12 characteristics for use in manufacturing various articles. Such materials are very much in
13 demand in the automobile and construction industries. To meet this demand, metal foam has
14 been proposed. Metal foam is generally formed by introducing a gas into a molten metal bath
15 to generate a foam on the surface thereof. Due to its high strength to weight ratio, aluminum
16 is a favoured metal to use in generating a foam, although other metals can also be used. The
17 foam is then removed and formed or cast into the desired shapes. Various methods have been
18 proposed for introducing the gas into the molten metal bath. Such methods include the use of
19 gas generating additives, blowing of air etc. With regard to the latter method, various
20 apparatus and systems are known for blowing a gas into the molten metal. Such apparatus
21 include nozzles, impellers and other such devices.

22
23 [0003] In US patent number 5,334,236, there is described a metal foam generating system
24 wherein air is introduced by means of a gas nozzle at the end of a supply tube or a hollow
25 rotating impeller having a plurality of openings through which the gas is passed. In both
26 cases, the tube or impeller is mounted on an angle into the metal bath through an opening.
27 There is no teaching in this patent as to how such opening is sealed to prevent the molten
28 metal from leaking. Further, the shafts used in forming the tubes or impellers are formed
29 from stainless steel due to the fact that they are immersed in molten metal. Nevertheless,
30 such shafts are known to become deteriorated after prolonged immersion in the molten metal
31 and must be replaced often. Another deficiency in these known gas introduction systems is
32 that since the shafts are provided in an angled manner into the molten metal bath, the length

1 of the shafts must be adjusted if the depth of the bath is increased. Apart from the drive
2 mechanism requirements of such an arrangement, it will be understood that the cost for each
3 shaft would also be greater. This, compounded with the need for constant replacement of the
4 shafts, results in a high cost of operation.

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6 [0004] In US application number 60/312,757, sharing a common inventor with the
7 present application, an improved metal foam generating and casting system is provided. In
8 this system, a metal foam is generated by introducing a gas into the bottom of the metal bath
9 to generate bubbles. The bubbles are then allowed to rise through a riser tube connected to a
10 die cavity. The bubbles then form a foam inside the cavity. After the cavity is filled, it is
11 allowed to cool and the formed metal foam article is retrieved. In this case, the generation of
12 bubbles at a specific location is desired. This reference provides a porous nozzle located at
13 the bottom of the molten metal bath, positioned generally directly under the riser tube.
14 Although such porous nozzle results in the desired foam generation, a rotating nozzle is
15 believed to improve the foam characteristics. However, the rotating nozzle shafts known in
16 the art have various disadvantages as described above. In this specific application, one other
17 disadvantage is that, with angled impeller shafts, it is often not possible to ensure that the
18 formed bubbles are introduced into the riser tube. Further, the above mentioned system
19 involves the pressurization of the foaming chamber. In such case an adequate seal around the
20 impeller is needed in order to prevent leakage. Such seal is difficult to establish in situations
21 where the impeller is introduced through the side of the molten metal bath.

22

23 [0005] Thus, there exists a need for an improved impeller system for generating metal
24 foam.

25

26 SUMMARY OF THE INVENTION

27 [0006] Thus, in one embodiment, the present invention provides a submerged gas
28 discharge impeller for supplying a gas to liquid within a container, said impeller comprising:
29 - a hollow shaft having at least one bore and a first end connected to a gas supply and
30 a second end extending into said liquid through an opening in the bottom of said container;
31 - the second end of said shaft including a gas discharge nozzle in fluid communication
32 with said bore;

- 1 - the shaft including a seal for preventing leakage of said fluid;
2 - a drive means for rotating the shaft about its longitudinal axis.

3

4 [0007] In another embodiment, the invention provides a system for discharging a gas
5 through a liquid, the system comprising:
6 - a container for said liquid, said container having a base with an opening;
7 - a hollow shaft having a first end connected to a gas supply and a second end
8 extending into said liquid through said opening in said container;
9 - a gas discharge nozzle connected to said second end of said shaft;
10 - a seal provided adjacent said opening in said container for preventing leakage of said
11 liquid;
12 - a motor connected to said shaft for rotating said shaft about its longitudinal axis.

13

14 [0008] In yet another embodiment, the invention provides a system for producing a metal
15 foam from a molten metal comprising:
16 - a bath containing said molten metal, said bath comprising a container with an
17 opening on the base thereof;
18 - a hollow, rotatable shaft extending generally vertically into said molten metal
19 through said opening, said shaft including a first end extending into said molten metal and a
20 second end connected to a gas supply;
21 - the first end of said shaft including a gas discharge nozzle submerged in said molten
22 metal;
23 - a seal located between said shaft and said opening for preventing passage of said
24 molten metal;
25 - a drive mechanism connected to said shaft for rotating said shaft about its
26 longitudinal axis.

27

28 BRIEF DESCRIPTION OF THE DRAWINGS

29 [0009] These and other features of the preferred embodiments of the invention will
30 become more apparent in the following detailed description in which reference is made to the
31 appended drawings wherein:

32 [0010] Figure 1 is a cross sectional elevation of a metal foam casting apparatus.

1 [0011] Figure 2 is a cross sectional elevation of a detail of molten metal bath illustrating
2 an impeller according to an embodiment of the present invention.

3 [0012] Figure 3 is a side view of a gas supply mechanism for the impeller of the
4 invention.

5

6 DESCRIPTION OF THE PREFERRED EMBODIMENTS

7 [0013] Figure 1 illustrates a metal foam casting system as taught in US application
8 number 60/312,757, described above, in which the present invention can be used. As
9 illustrated, the casting system includes a die 36 having a die cavity 38, which is fluidly
10 connected to a riser tube 39. The riser tube 39 extends into a bath 32 containing a molten
11 metal 34. The bath 32 also includes, at the base thereof, a porous plug, or nozzle, 44. A gas
12 supply line 42, connected to the nozzle 44, introduces a gas through the nozzle 44, into the
13 molten metal 34. Such gas leads to the formation of bubbles 46 which, due to their
14 buoyancy, preferentially rise in the direction shown by the arrow C. As can be seen, by
15 positioning the riser tube 39 generally directly over the nozzle 44, the bubbles are caused to
16 enter such tube and rise to form a metal foam. As will be appreciated the opening of the tube
17 39 may be provided with a funnel shaped end to assist in collecting the formed bubbles. The
18 foam is, thereby, allowed to enter and fill the die cavity 38. As will be understood by persons
19 skilled in the art, once the die cavity is filled with the metal foam, the die can be cooled to
20 solidify the foam and, subsequently, remove the formed foam article.

21

22 [0014] Figure 2 illustrates a rotating gas supply impeller for use, in one example, as an
23 alternative to the stationary porous nozzle of the metal foam casting system described above
24 and as illustrated in Figure 1.

25

26 [0015] The rotating impeller according to one embodiment of the invention is shown
27 generally at 100 in Figure 2. The impeller includes a hollow shaft 102 that extends generally
28 vertically into the base 104 of the molten metal bath (not shown). As is commonly known in
29 the art, the bath, including the base 104, is provided with a refractory or insulating material
30 105 that is capable of withstanding the temperatures of the molten metal. A first, bottom end
31 106 of the shaft 102 provides and exposed opening 108 into the hollow bore 110 of the shaft

1 102. Air is introduced into the bore 110 of the shaft 102 by connecting a gas supply line
2 (discussed further below) to the opening 108.

3

4 [0016] Turning briefly to Figure 3, an example of a gas supply arrangement is illustrated.
5 As shown, the shaft 102 includes a threaded portion (not shown) on the interior wall of the
6 bore 110. A rotary union 160 includes a threaded connector 162 having a thread that is
7 complementary to that of the bore 110. The rotary union 160 is secured to the shaft 102 by
8 screwing the connector 162 into to the bore 110. The rotary union 160 includes a rotating
9 section 164 and a stationary section 166. The means of linking sections 164 and 166 together
10 is commonly known and, indeed, the rotary union 160 itself is commercially available. A gas
11 supply port 168 is provided on stationary section 166. A gas supply line 170 is then attached
12 to the supply port 168. Although preferred gas supply system has been described, various
13 other methods of providing a gas supply to the shaft 102 will be apparent to persons skilled in
14 the art.

15

16 [0017] Returning to Figure 2, on the second, top end 112 of the shaft 102, there is
17 attached a gas outlet nozzle 114. The top end 112 of the shaft 102 extends into the molten
18 metal bath through an opening 116, which extends through the base 104 and refractory
19 material 105. A support 118 having a central bore 120 is provided in the opening 116 in the
20 base 104. The shaft 102 extends through the central bore 120 of the support 118, with the
21 central bore 120 being dimensioned to allow free rotation of the shaft 102. The support 118
22 includes a generally conical upper portion 122, which includes an annular shoulder 124 that
23 bears against a portion the inner surface 126 of the base 104, such portion being adjacent to
24 the opening 116. The support 118 also includes a generally cylindrical body 117, through
25 which extends the bore 120, the body 117 preferably extending through the opening 116.
26 The outer diameter of the body 117 is preferably dimensioned to provide a snug fit within the
27 opening 116. As indicated above, the upper portion 120 of the support 118 has a generally
28 conical structure. Such structure aids in directing molten metal away from the shaft 102.
29 Although the support 118 and the opening 116 are described in terms of preferred structural
30 configurations, it will be understood by persons skilled in the art that various other
31 geometries are possible within the scope of the present invention as described herein. It will
32 also be understood that the support 118 is preferably made from a material that is capable of

1 withstanding the temperature of the molten metal. For example, suitable materials include
2 alumina silicate, graphite or ceramics.

3

4 [0018] The central bore 120 of support 118 includes an upper region 121, at the top end
5 of the support 118, which has a larger diameter than that of the bore 120. Such widened
6 diameter provides a ledge 128, which supports a seal or bushing 130. The bushing 130 has a
7 generally cylindrical outer wall 132 that corresponds generally to the diameter of the upper
8 region 121 of the support 118. In the preferred embodiment, the bushing 130 is maintained in
9 position within the upper region 121 by frictional contact between its outer wall 132 and the
10 inner wall of the upper region 121. Further, such arrangement ensures a tight seal between
11 the bushing 130 and the support 118. In the preferred embodiment, the bushing 130 is made
12 of graphite to withstand the temperatures of the molten metal to which it is exposed.
13 However, other materials will be apparent to persons skilled in the art such as ceramics,
14 metals, or composites. Some examples of possible materials for the bushing 130 include,
15 *inter alia*, graphite, titanium diboride, tungsten, alumina, zirconium oxide (ZrO_2), silicon
16 carbide, silicon nitrate, boron nitrate, titanium carbide and tungsten carbide.

17

18 [0019] In another embodiment, the support 118 can be integrally formed with the seal or
19 bushing 130. However, it will be understood that a separate seal is preferred so as to
20 facilitate replacement as the seal 130 wears out. It will also be understood that for forming an
21 optimal seal, the underside of the nozzle 114 should be square with the upper contacting
22 surface of the seal or bushing 130.

23

24 [0020] In a preferred embodiment, the material chosen for the seal or bushing 130 is non-
25 wetted by the molten metal. Similarly, the impeller or parts thereof is also made of a non-
26 wetted material. In another embodiment, the elements in contact with the molten metal, i.e.
27 the seal bushing 130, the support 118, the nozzle 114, and any other parts of the impeller,
28 may be coated with a protective material that resists wetting by the molten metal and/or to
29 seal the apparatus to prevent leakage.

30

31 [0021] The bushing 130 also includes a central bore 134, which accommodates the upper
32 end of the shaft 102 and allow for rotation of the shaft therein. The clearance between the

1 outer diameter of the shaft 102 and the bore 134 of the bushing 130 is preferably maintained
2 as minimal as possible so as to provide a sealing arrangement there-between. In this manner,
3 and with the seal between the bushing 130 and the support 118, leakage of molten metal
4 within the bath is prevented.

5

6 [0022] The gas discharge nozzle 114 preferably comprises a generally cylindrical body
7 secured to the top end of the shaft. In the preferred embodiment, the body of the nozzle 114
8 comprises a plurality of fins 115 extending radially from the central axis of the body. The
9 nozzle 114 also includes a central opening 136 in fluid communication with the central bore
10 108 of the shaft 102. In the preferred embodiment, the opening 136 does not extend through
11 the entire body of the nozzle 114 and, instead, the body of the nozzle 114 is provided with
12 one or more, and more preferably, a plurality of gas discharge vents 138 extending through
13 the fins 115. The vents 138 radiate from, and are in fluid communication with, the opening
14 136 of the nozzle 114. The vents 138 open into the molten metal bath so as to discharge the
15 gas supplied through the shaft 102 into the molten metal. By securing the nozzle 114 to the
16 shaft 102, it will be understood that rotation of the shaft 102 also results in the rotation of the
17 nozzle. In the preferred embodiment, the bottom surface of the nozzle 114 abuts the top
18 surface of the bushing 130 so as to establish a sealing arrangement there-between.

19

20 [0023] The shaft 102 extends through an opening in a stationary support 140 located
21 below the bath. The support 140 preferably includes a bearing 142 having a central bore 144
22 that is greater in diameter than that of the shaft 102. The bore 144 is preferably provided
23 with a bushing 146 through which is passed the shaft 102. It will be understood that the shaft
24 102 is rotatably accommodated within the bushing 146. One of the purposes of the bearing
25 142 is, as will be understood, to support and stabilize the shaft 102 while it is rotated. The
26 bearing 142 is preferably also provided with a washer 148 on the bottom thereof, through
27 which is passed the shaft 102. The purpose of the washer 148 is described below.

28

29 [0024] At the bottom end 106 of the shaft 102, there is provided a collar 150, secured to
30 the shaft. Between the collar 150 and the washer 148, there is provided a spring 152, the
31 spring being in a compressed state. As will be understood, the spring, being provided in this
32 manner, exerts a force bearing against the washer 148 and the collar 150, causing the washer

1 and the collar to be forced away from each other. This force will extend along the length of
2 the shaft 102 thereby causing the bottom surface of the nozzle 114 to bear against the top
3 surface of the bushing 130, thereby serving to strengthen the seal between the nozzle and the
4 bushing to prevent leakage of molten metal from the bath. It will also be understood that
5 such force will also ensure that the support 118 is pressed against the inner surface of the bath
6 to ensure a seal there-between as well. It will be appreciated, however, that the primary
7 reason for applying a force by means of the spring 152 is to seal the nozzle against the
8 bushing. Although the use of a spring 152 is a preferred method of achieving the desired
9 seal, it will be understood that any other means may also be employed. For example, the
10 shaft 102 may be attached to any other force applying means to achieve the desired result.
11 Alternatively, the weight of the shaft and associated elements may be sufficient to provide the
12 necessary sealing force.

13

14 [0025] The present invention envisages various means of rotating the shaft 102. In one
15 embodiment, the shaft 102 is provided with a pulley 154, secured to the shaft 102 in a
16 location along the length thereof. The pulley 154 translates a drive force applied thereto into
17 axial rotation of the shaft 102. As is known in the art, the pulley 154 is adapted to engage a
18 drive belt that is connected to a drive motor (not shown). In another embodiment, the pulley
19 154 may be replaced with a sprocket that engages a cooperating sprocket on a drive shaft of a
20 motor. The choice drive means for axially rotating the shaft 102 will depend upon the drive
21 mechanism being used. It will also be understood that locating the drive means (for example
22 the pulley 154) away from the bottom end 106 of the shaft 102 is preferred so as not to
23 interfere with the gas supply line feeding the bore 108.

24

25 [0026] In the preferred embodiment, a further bearing 156 is provided on the underside of
26 the base 104 of the bath. The bearing 156 can be, for example, of the same structure as
27 bearing 142 described above. It will be understood that the purpose of the bearing 156 is to
28 support and stabilize the shaft 102 while it is rotated. It will also be understood that in other
29 embodiments of the invention, the bearing 156 may not be needed if the shaft 102 is able to
30 support itself. As shown, in the preferred embodiment of the invention, the bearing 156 is
31 also provided with a bushing 157 similar to bushing 146. It will also be appreciated that any
32 number of bearings or bushings can be used depending upon the needs of the apparatus.

1

2 [0027] As described above, an impeller according to the present invention improves the
3 dispersal of the gas discharged within the molten metal. Also, the impeller of the invention,
4 by minimizing or eliminating the length of the shaft exposed to the molten metal, avoids
5 damage thereto as described above as well as other deleterious effects of having a rotating
6 shaft within the fluid molten metal. Also, by providing a means of discharging gas directly
7 from the bottom of the bath, the desired vertical rise of the gas bubbles is achieved.

8

9 [0028] In the above described embodiments, a system having a single impeller shaft and
10 gas discharge nozzle has been described. However, the invention also contemplates other
11 systems wherein several impellers and nozzles are employed. As will be apparent to persons
12 skilled in the art, more than one impeller and nozzle combination may be more efficient when
13 large diameter riser tubes 39 are used.

14

15 [0029] The present invention has been described in terms of its use in a metal foam
16 casting system. However, it will be appreciated that this is only one possible use of the
17 invention and that various other uses are within the scope thereof. Although impeller speeds
18 of around 4500 rpm are known in art of metal foam generation, any other desired speed
19 would, of course, be possible.

20

21 [0030] Although the invention has been described with reference to certain specific
22 embodiments, various modifications thereof will be apparent to those skilled in the art
23 without departing from the spirit and scope of the invention as outlined in the claims
24 appended hereto.